



NATIONAL UNIVERSITY OF SCIENCE AND TECHNOLOGY

FACULTY OF INDUSTRIAL TECHNOLOGY

DEPARTMENT OF CIVIL AND WATER ENGINEERING

STRUCTURAL ANALYSIS I

TCW 3102

Supplementary Examination Paper

July 2015

This examination paper consists of 7 pages

Time Allowed: 3 hours

Total Marks: 100

Special Requirements:

Examiner's Name: Miss Diana Makweche

INSTRUCTIONS

1. Answer any four (4) questions
2. Each question carries 25 marks
3. Use of calculators is permissible

MARK ALLOCATION

QUESTION	MARKS
1.	25
2.	25
3.	25
4.	25
5.	25
TOTAL	100

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TCW 3102

QUESTION 1

(a) Qualitatively analyse the structures in Figure Q1A for:

- (i) deflected shape
- (ii) reactions
- (iii) shear force diagram
- (iv) bending moment diagram

Clearly indicate points of interest (i.e. inflexion points, regions of zero slope, straight portions/members, zones of tension, etc)

[10]

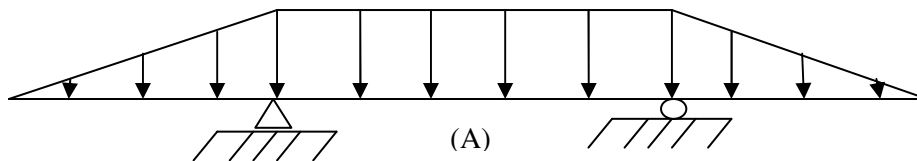


Figure Q1A

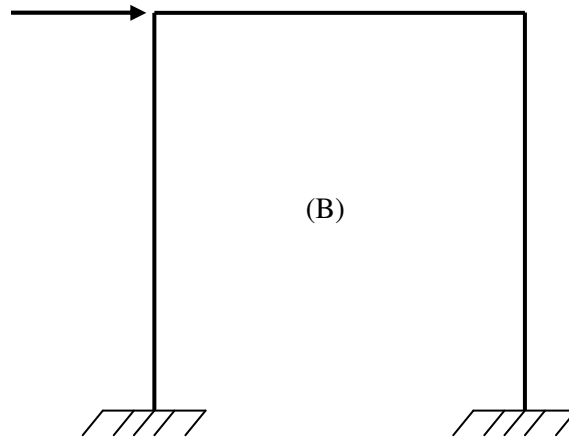


Figure Q1B

(b) In Figure Q1B above, if the horizontal force is 20kN. Find the principal values of the bending moment and shear force diagrams using any method.

[15]

QUESTION 2

(a) Describe the application of influence lines in structural analysis. [3]

(b) Use the Muller-Breslau Method to sketch the influence lines for the reaction at A and shear at B of Figure Q2A. (Show relative values) [6]

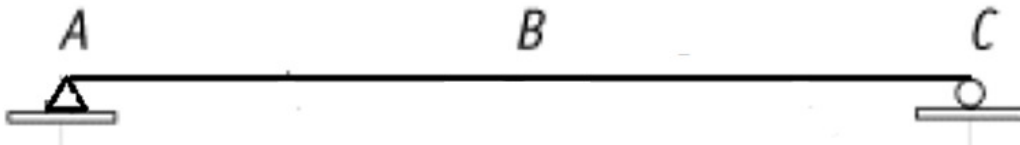


Figure Q2A

(b) For the structure in Figure Q2B, find the maximum positive shear at B due to the system of loads as it moves from right to left. [16]

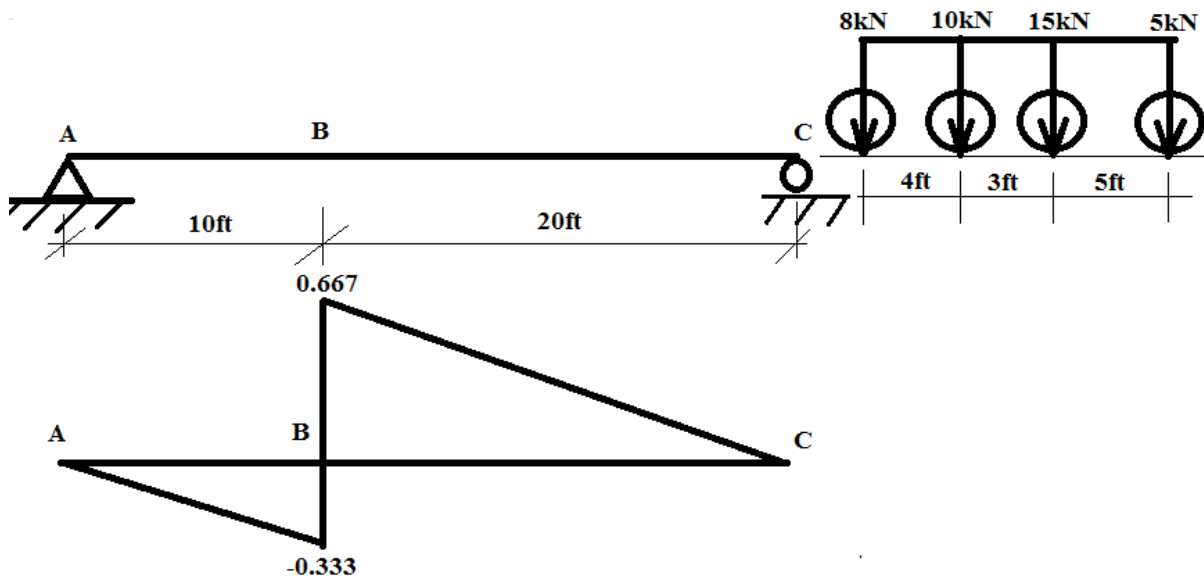


Figure Q2B

QUESTION 3

Analyse the beam shown in Figure Q3. The beam is pinned at A and sits on a roller support at B. Support C is fixed. Construct the bending moment diagram. [25]

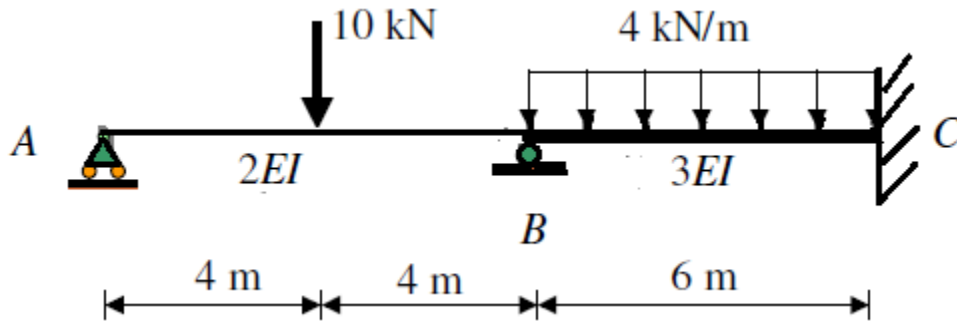


Figure Q3

QUESTION 4

Analyse the frame in Figure below. Draw the bending moment diagram. EI is constant. [25]

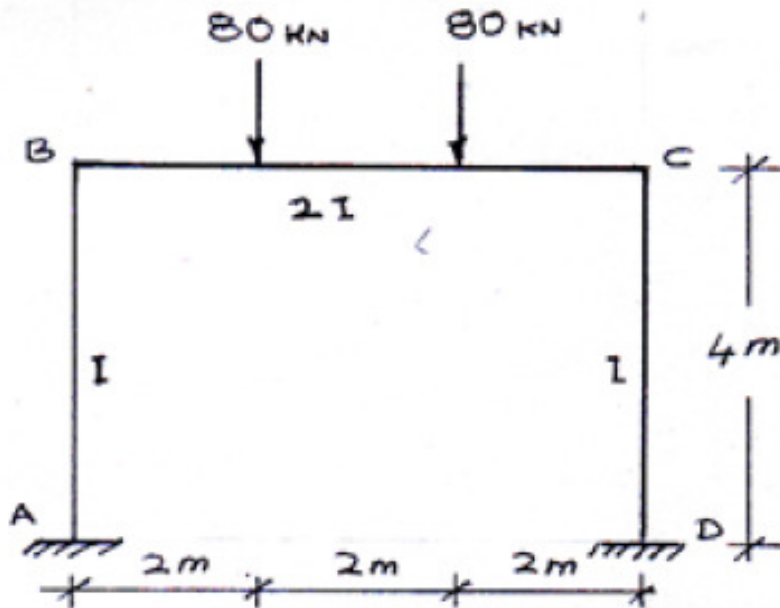
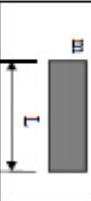


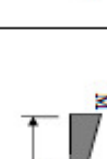

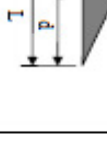

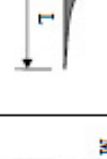


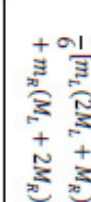

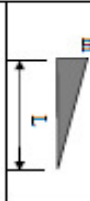
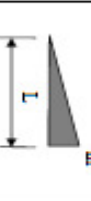
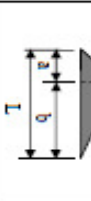

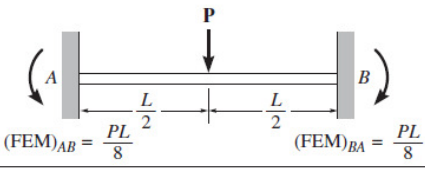
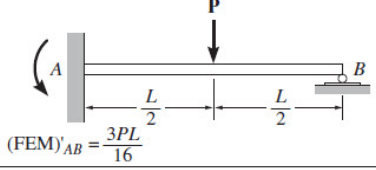
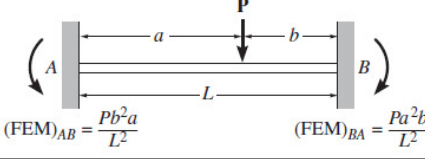
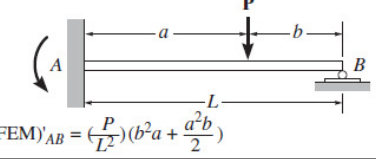
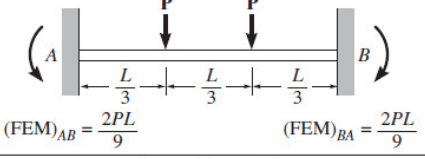
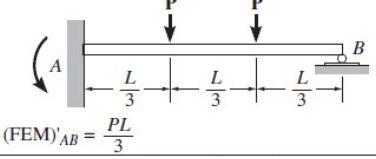
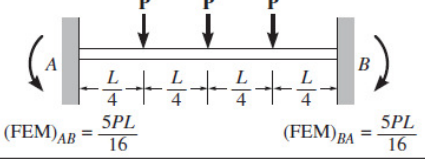
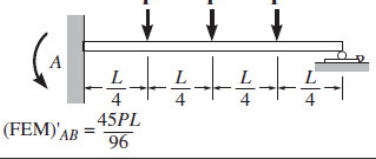
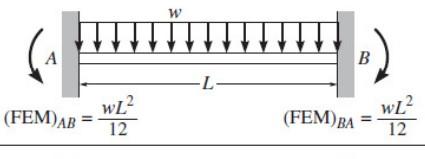
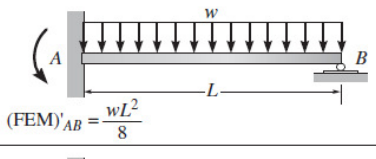
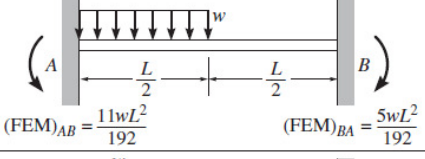
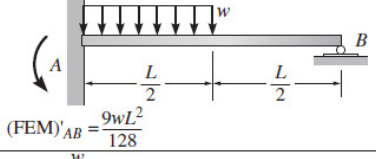
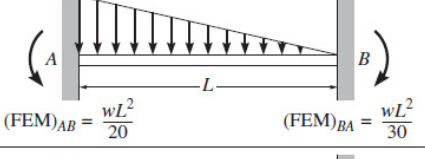
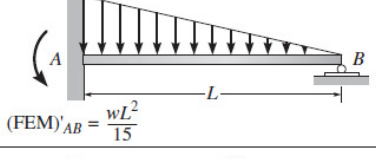
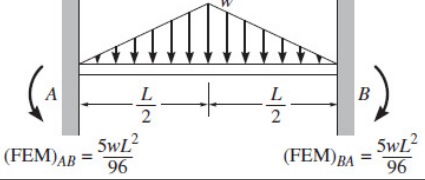
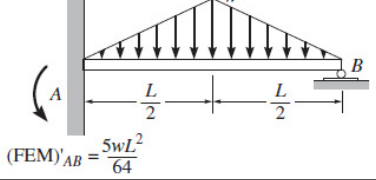
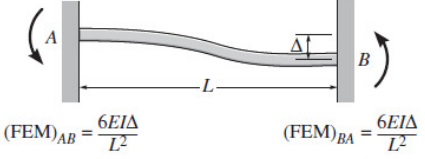
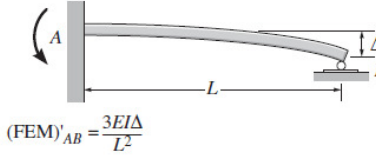


Figure Q4

To Evaluate Product Integrals of the Form: $\int_0^L mMdx$

		LmM		$\frac{L}{2}m(M_L + M_R)$		$\frac{L}{2}mM$		$\frac{L}{2}mM$		$\frac{L}{2}mM$		$\frac{L}{2}mM$		$\frac{L}{3}mM$		$\frac{2L}{3}mM$
	$\frac{L}{2}(m_L + m_R)M$	$\frac{L}{6}[m_L(2M_L + M_R) + m_R(M_L + 2M_R)]$		$\frac{L}{6}(2m_L + m_R)M$		$\frac{L}{6}m\left(1 + \frac{d}{L}\right)M$	$\frac{L}{6}[m_L\left(1 + \frac{d}{L}\right) + m_R\left(1 + \frac{c}{L}\right)]M$	$\frac{L}{4}(m_L + m_R)M$	$\frac{L}{12}(3m_L + m_R)M$	$\frac{L}{12}(5m_L + 3m_R)M$						
	$\frac{L}{2}mM$	$\frac{L}{6}m(2M_L + M_R)$	$\frac{L}{3}mM$	$\frac{L}{6}m\left(1 + \frac{d}{L}\right)M$	$\frac{L}{4}mM$	$\frac{L}{4}mM$	$\frac{L}{4}mM$	$\frac{L}{4}mM$	$\frac{L}{4}mM$	$\frac{5L}{12}mM$						
	$\frac{L}{2}mM$	$\frac{L}{6}m(M_L + 2M_R)$	$\frac{L}{6}mM$	$\frac{L}{6}m\left(1 + \frac{c}{L}\right)M$	$\frac{L}{4}mM$	$\frac{L}{12}mM$	$\frac{L}{4}mM$	$\frac{L}{12}m\left(1 + \frac{b}{L} + \frac{b^2}{L^2}\right)M$	$\frac{L}{4}mM$	$\frac{L}{4}mM$						
	$\frac{L}{2}mM$	$\frac{L}{6}m\left[M_L\left(1 + \frac{b}{L}\right) + M_R\left(1 + \frac{a}{L}\right)\right]$	$\frac{L}{6}m\left(1 + \frac{b}{L}\right)M$	$\frac{L}{6}m\left(1 + \frac{c}{L}\right)M$	$\frac{L}{4}mM$	$\frac{L}{12}m\left(1 + \frac{b}{L} + \frac{b^2}{L^2}\right)M$	$\frac{L}{12}m\left(5 - \frac{a}{L} - \frac{a^2}{L^2}\right)M$									
	$\frac{L}{2}mM$	$\frac{L}{4}m(M_L + M_R)$	$\frac{L}{4}mM$	$\frac{L}{6}m\left(1 + \frac{c}{L}\right)M$	$\frac{L}{3}mM$	$\frac{7L}{48}mM$	$\frac{17L}{48}mM$									

Fixed End Moments

 <p> $(FEM)_{AB} = \frac{PL}{8}$ $(FEM)_{BA} = \frac{PL}{8}$ </p>	 <p> $(FEM)'_{AB} = \frac{3PL}{16}$ </p>
 <p> $(FEM)_{AB} = \frac{Pb^2a}{L^2}$ $(FEM)_{BA} = \frac{Pa^2b}{L^2}$ </p>	 <p> $(FEM)'_{AB} = \left(\frac{P}{L^2}\right)(b^2a + \frac{a^2b}{2})$ </p>
 <p> $(FEM)_{AB} = \frac{2PL}{9}$ $(FEM)_{BA} = \frac{2PL}{9}$ </p>	 <p> $(FEM)'_{AB} = \frac{PL}{3}$ </p>
 <p> $(FEM)_{AB} = \frac{5PL}{16}$ $(FEM)_{BA} = \frac{5PL}{16}$ </p>	 <p> $(FEM)'_{AB} = \frac{45PL}{96}$ </p>
 <p> $(FEM)_{AB} = \frac{wL^2}{12}$ $(FEM)_{BA} = \frac{wL^2}{12}$ </p>	 <p> $(FEM)'_{AB} = \frac{wL^2}{8}$ </p>
 <p> $(FEM)_{AB} = \frac{11wL^2}{192}$ $(FEM)_{BA} = \frac{5wL^2}{192}$ </p>	 <p> $(FEM)'_{AB} = \frac{9wL^2}{128}$ </p>
 <p> $(FEM)_{AB} = \frac{wL^2}{20}$ $(FEM)_{BA} = \frac{wL^2}{30}$ </p>	 <p> $(FEM)'_{AB} = \frac{wL^2}{15}$ </p>
 <p> $(FEM)_{AB} = \frac{5wL^2}{96}$ $(FEM)_{BA} = \frac{5wL^2}{96}$ </p>	 <p> $(FEM)'_{AB} = \frac{5wL^2}{64}$ </p>
 <p> $(FEM)_{AB} = \frac{6EI\Delta}{L^2}$ $(FEM)_{BA} = \frac{6EI\Delta}{L^2}$ </p>	 <p> $(FEM)'_{AB} = \frac{3EI\Delta}{L^2}$ </p>

Slope Deflection Equations

$$M_N = 2Ek(2\theta_N + \theta_F - 3\psi) + (FEM)_N$$

$$M_N = 3Ek(\theta_N - \psi) + (FEM)_N$$